

Control Volume: Region of space through which mass flows in and out

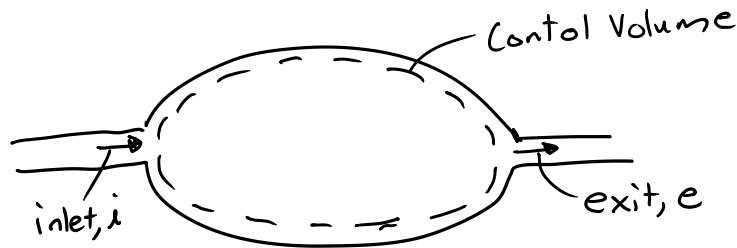
Conservation of Mass for a Control Volume

$$\frac{dm_{cv}}{dt} = \dot{m}_i - \dot{m}_e$$

time rate

inlet exit

time rate of change of mass contained within the control volume

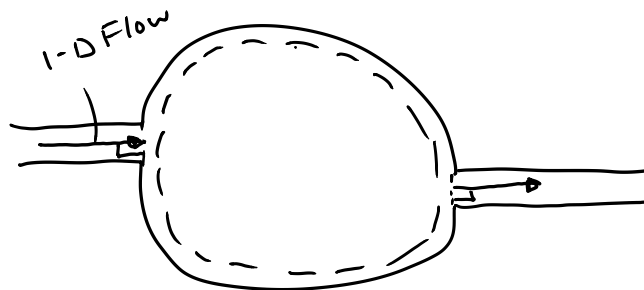


For multiple inlets and exits

$$\frac{dm_{cv}}{dt} = \sum \dot{m}_i - \sum \dot{m}_e$$

One-Dimensional Flow

- Flow is normal to the boundary at all locations where mass enters and exits the control volume
- All properties are uniform with position over each inlet and exit area (no change along the cross-section)



$$\dot{m} = \rho A v \quad (\text{1-D Flow}) \quad \rho = \frac{1}{v}$$

$$\dot{m} = \frac{(A v)}{v} \quad \text{Volumetric Flow Rate}$$

$$\frac{dm_{cv}}{dt} = \sum \left(\frac{A_i v_i}{v_i} \right) - \sum \left(\frac{A_e v_e}{v_e} \right)$$

Steady-State Flow

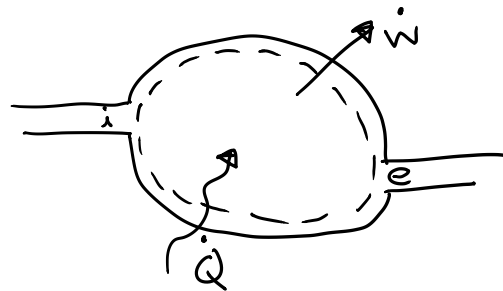
$$\frac{dm_{cv}}{dt} = 0$$

$$\sum \dot{m}_i = \sum \dot{m}_e \quad (\text{Steady-state})$$

Conservation of Energy for a Control Volume

Energy at the Inlet

$$\begin{aligned} \dot{U}_i + \frac{1}{2} \dot{m}_i v_i^2 + \dot{m}_i g z_i \\ = \dot{m} u_i + \frac{1}{2} \dot{m}_i v_i^2 + \dot{m}_i g z_i \\ = \dot{m}_i \left(u_i + \frac{1}{2} v_i^2 + g z_i \right) \end{aligned}$$



Rate of Energy Change

$$\dot{m}_i \left(u_i + \frac{1}{2} v_i^2 + g z_i \right)$$

For the Exit

$$\dot{m}_e \left(u_e + \frac{1}{2} v_e^2 + g z_e \right)$$

Energy Rate Balance

$$\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + \dot{m}_i \left(u_i + \frac{1}{2} v_i^2 + g z_i \right) - \dot{m}_e \left(u_e + \frac{1}{2} v_e^2 + g z_e \right)$$

Work Components for the Control Volume

- 1.) Fluid pressure at the inlets and exits
- 2.) Everything else

Fluid Pressure

$$W_1 = \text{Force} \times \text{Distance}$$

$$= (pA) \times s$$

$$\dot{W}_1 = pA \left(\frac{ds}{dt} \right)$$

$$\dot{W}_1 = pA \underbrace{v}_{\dot{V}} \quad \dot{m} = \frac{A \dot{V}}{v} \quad A v = \dot{m} v$$

$$\dot{W}_1 = p \dot{m} v$$

Everything Else

$$\dot{W}_2 = \dot{W}_{cv}$$

$$\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - (\dot{W}_{cv} - p_a \dot{m}_i v_i + p \dot{m}_e v_e) + \dot{m}_i \left(u_i + \frac{1}{2} v_i^2 + g z_i \right) - \dot{m}_e \left(u_e + \frac{1}{2} v_e^2 + g z_e \right)$$

$$\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + \dot{m}_i \underbrace{\left(u_i + p v_i + \frac{1}{2} v_i^2 + g z_i \right)}_{h_i} - \dot{m}_e \underbrace{\left(u_e + p v_e + \frac{1}{2} v_e^2 + g z_e \right)}_{h_e}$$

$$\boxed{\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + \dot{m} \left(h_i + \frac{1}{2} v_i^2 + g z_i \right) - \dot{m}_e \left(h_e + \frac{1}{2} v_e^2 + g z_e \right)}$$

Steady State

$$\frac{dE_{cv}}{dt} = 0$$

$$\boxed{0 = \dot{Q}_{cv} - \dot{W}_{cv} + \dot{m}_i \left(h_i + \frac{1}{2} v_i^2 + g z_i \right) - \dot{m}_e \left(h_e + \frac{1}{2} v_e^2 + g z_e \right)}$$

One Inlet, One Exit

$$\dot{m}_i = \dot{m}_e = \dot{m}$$

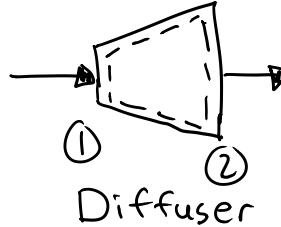
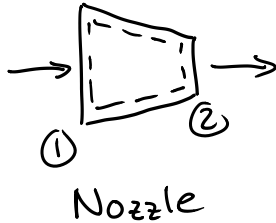
① \Rightarrow Inlet
② \Rightarrow Exit

$$\boxed{0 = \dot{Q}_{cv} - \dot{W}_{cv} + \dot{m} \left[(h_1 - h_2) + \frac{1}{2} (v_1^2 - v_2^2) + g(z_1 - z_2) \right]}$$

Control Volume Components

Nozzles and Diffusers

- Varying cross-sectional area
- Nozzle: Area decreases in the direction of the flow (pressure and velocity increases)
- Diffuser: Area increases in the direction of the flow (pressure and velocity decreases)

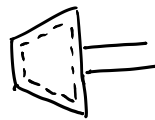


Typically

$$\dot{W}_{cv} = \Delta PE = 0$$

Turbines

- Gas or liquid passing through a set of blades connected to a rotating shaft

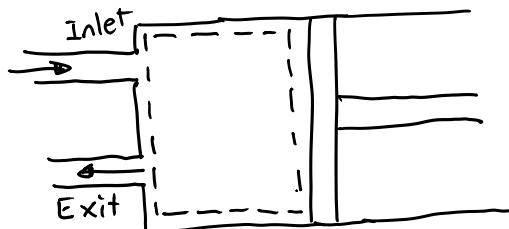


Typically

$\dot{Q}_{cv}, \Delta KE, \Delta PE$ (except hydroelectric) are small with respect to Work and Enthalpy

Compressors and Pumps

- Compressor: Work is done on a gas passing through the compressor to raise the pressure
- Pumps: Liquids



Typically

Same as turbines but there could be heat transfer

Heat Exchangers

- Energy is transferred by heat between fluids at different temperatures

Typically

- If the heat exchanger is modeled as a whole (both fluids), then \dot{Q}_{cv} is small. If not, \dot{Q}_{cv} is significant
- \dot{W}_{cv} , ΔKE , ΔPE are small

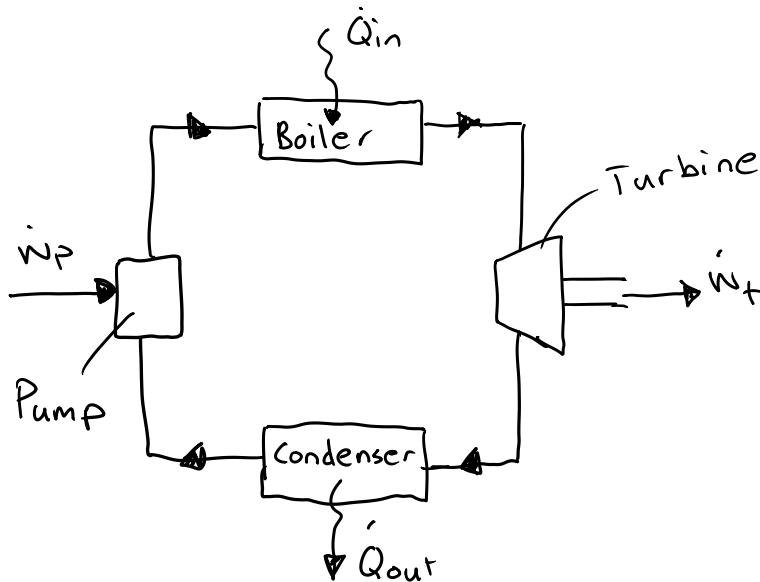
Throttling Devices

- Restriction in the line to reduce pressure (partially opened valve or a porous plug)

Typically

- \dot{Q}_{cv} , \dot{W}_{cv} , ΔPE are small
- $\Delta KE \Rightarrow$ High velocity at the restriction, but much smaller further away

System Integration (Vapor Power Plant)



ENGR 2240 – Thermodynamics
Section 3: Control Volume Analysis